



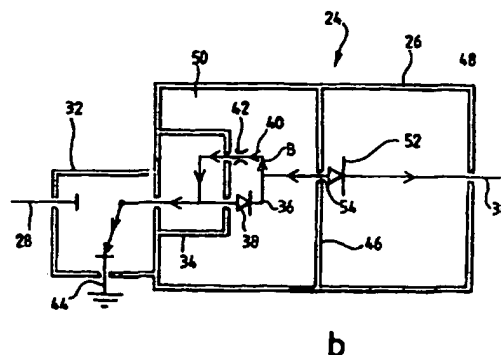
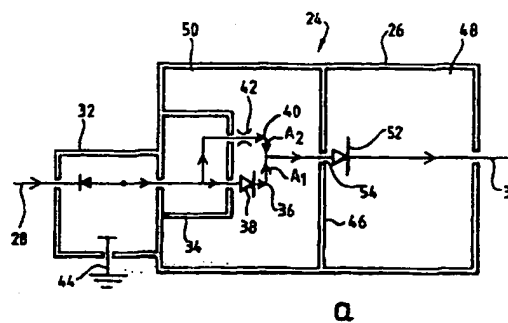
INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(51) International Patent Classification ⁶ : B60T 17/00	A1	(11) International Publication Number: WO 98/12089 (43) International Publication Date: 26 March 1998 (26.03.98)
<p>(21) International Application Number: PCT/GB97/02528</p> <p>(22) International Filing Date: 18 September 1997 (18.09.97)</p> <p>(30) Priority Data: 9619535.9 19 September 1996 (19.09.96) GB</p> <p>(71) Applicant (for all designated States except US): WABCO AUTOMOTIVE U.K. LIMITED [GB/GB]; Beacon Works, Texas Street, Morley, Leeds LS27 0HQ (GB).</p> <p>(72) Inventor; and (75) Inventor/Applicant (for US only): THOMAS, Barry, David [GB/GB]; 18 Moorside View, Drighlington, Bradford BD11 1HN (GB).</p> <p>(74) Agent: CHETTLE, Adrian, John; Withers & Rogers, 4 Dyer's Buildings, Holborn, London EC1N 2JT (GB).</p>	<p>(81) Designated States: BR, US, European patent (AT, BE, CH, DE, DK, ES, FI, FR, GB, GR, IE, IT, LU, MC, NL, PT, SE).</p> <p>Published With international search report.</p>	

(54) Title: AIR BRAKING SYSTEM COMPONENT

(57) Abstract

An air dryer (24) for a vehicle air braking system comprises a container (26) having first, second and third compartments (34, 50, 48), the first compartment (34, 34a) having an upstream inlet (28) for connection to a source of air under pressure, a vent (44) for connection to the exterior of the container (26) and a desiccant therein, the second compartment (50) comprising a chamber having an inlet (36, 36a) downstream of the first compartment (34, 34a) with non-return means (38, 38a) therein, a fluid connection (40, 40a) to the first compartment (34, 34a) having a restrictor (42, 42a) therein, and the third compartment (48) comprising a chamber having an inlet (54) downstream of the second compartment (50) with non-return means (52) therein and an outlet (30) for connection to a fluid pressure using device.



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AIR BRAKING SYSTEM COMPONENT

This invention relates to a component for an air braking system, and particularly to a combined air dryer/reservoir for a vehicle air braking system.

5 Air braking systems are typically found on heavy commercial vehicles. They usually comprise a compressor, a demand valve, one or more reservoirs and plurality of actuators for the vehicle wheel brakes. One common addition to an air braking system is an air dryer. An air dryer
10 typically comprises a canister of desiccant downstream of the compressor and through which air is passed to the system. In time, the desiccant becomes less effective, as moisture is adsorbed, and it is necessary to regenerate the desiccant for example by reverse flow of dry air and
15 exhausting the resultant moist air through a vent valve to atmosphere. Regeneration for a single chamber reservoir is carried out when the compressor is off-load using dry air from the system volume or from a dedicated purge tank downstream of the air dryer. Alternatively, parallel air
20 dryers can be provided where little off-load time is available; regeneration for either one of the air dryers is carried out when the compressor is on-load and connected through the other air dryer; dry air is supplied to the 'wet' air dryer from a secondary downstream reservoir.

25 A typical air system lay out with a single air dryer is shown schematically in Figure 1. The system comprises, in series, a compressor 10, an air dryer 12, a sensing tank

14 and reservoirs 16,18 downstream of non-return valves 20,22 respectively. A pressure sensor in the tank 14 causes the compressor to be brought on load or off load according to the sensor state, and consequently to switch the air
5 dryer inlet between pressure feed from the compressor and exhaust whereby dry air flows back through the desiccant chamber. The system shown requires rather a lot of piping to connect the individual components; it is not particularly compact and is consequently disadvantageous where space is
10 limited.

It is an object of the invention to provide an improved component including air drying means for an air braking system.

Accordingly there is provided a component of an air
15 braking system comprising a container having first and second compartments, the first compartment having an upstream inlet for connection to a source of air under pressure, a vent for connection to the exterior of the container and a desiccant therein, the second compartment
20 comprising a chamber having a downstream inlet from the first compartment with non-return means therein, a return connection to the first compartment having a fluid restrictor therein, and an outlet for connection to an air braking system.

25 In that way the desiccant compartment is located next to the air reservoir constituted by the second compartment, which results in a much more compact system in which the

external shape of the component can be adapted to the available space. The component of the invention is suitable for air braking systems in which the desiccant is regenerated when the compressor is off load but working and systems where the desiccant is regenerated whilst the compressor is deactivated. Furthermore the component of this invention can be used in systems which have significant compressor off load time, and systems where off load time is restricted and thus switching between alternative desiccant beds is required.

Preferably the component includes a control valve to regulate the passage of air through said upstream inlet and vent, and the control valve may be mounted to the container. The control valve may be controlled by a conventional demand valve signal such that regeneration of the desiccant is initiated at a suitable time.

In an alternative embodiment the component may include a third compartment having an upstream inlet for connection to a source of air under pressure, a vent for connection to the exterior of the container and a desiccant therein, the second compartment also having a downstream inlet from the third compartment with non-return means therein, and a return connection to the third compartment having a fluid restrictor therein. Preferably a control valve is provided to regulate the passage of air through the upstream inlet and vent of said third compartment. A common control valve may be provided to regulate the passage of air through said

upstream inlets and vents of said first and third compartments. In that way air can be directed between the first and third compartments and thus permit a relatively wet desiccant compartment to be regenerated with dry air whilst air under pressure is supplied through the relatively dry desiccant compartment.

Preferably the control valve is adapted to selectively direct air to the upstream inlet of the first compartment and to the upstream inlet of the third compartment. The control valve may be adapted to direct air between said upstream inlets according to a timer, a sensing device adapted to monitor the degree of moisture in said first and third compartments, or any other suitable sensor.

Preferably the compartments are defined by a partition within the container and the first compartment may project into the second compartment. Where two desiccant chambers are provided, both may project into the air reservoir constituted by the second compartment.

There may be provided a further compartment comprising a chamber connected to the outlet of said second compartment, non-return means being provided at said outlet and said further compartment being provided with a supply connection to an air braking system. This arrangement places successive air reservoirs in series, the upstream reservoir constituting a relatively small reservoir for desiccant drying purposes, and the downstream reservoir constituting a main supply tank for the braking system.

The component is preferably modular, most preferably in the form of a canister for installation into an air braking system. In such a case the inlets and outlets may have appropriate pressure fittings. The component may for example be screw threaded onto a valve block in a manner similar to that of an oil filter.

The desiccant may be removable from the first and third compartments.

The invention also encompasses an air braking system, particularly a vehicle air braking system, having a component according to the invention.

Embodiments of components in accordance with the invention will now be described in detail, by way of example, and with reference to the accompanying drawings, in which:

Figure 1 is a schematic cross-section through a known air system.

Figure 2 is a schematic cross-section through a first component in accordance with the invention,

Figures 2a and 2b are schematic representations of air flow through the component of Fig. 2 in drying and regenerating states respectively,

Figure 3 is a schematic cross-section through a second

component in accordance with the invention,

Figures 3a and 3b are schematic representations of air flow through the component of Fig. 3 showing drying of compartment 34a and 34b respectively,

5 Figure 4 is a longitudinal cross-section of a third component in accordance with the present invention,

Figures 4a and 4b are schematic representations of air flow through the component of Fig. 4 in drying and regenerating states respectively,

10 Figure 5 is a schematic cross-section through a fourth component in accordance with the present invention, and

Figures 5a and 5b are schematic representations of air flow through the component of Figure 5 showing drying of compartments 34a and 34b respectively.

15

In Figure 2, a component generally indicated at 24 comprises an airtight container 26 having an inlet 28 and an outlet 30. The inlet 28 is closed by a control valve 32. The outlet 30 is normally closed but is openable in use by
20 a demand valve of an air braking system (not shown).

A desiccant compartment 34 is located downstream of the inlet 28 within the container 26. The desiccant

compartment 34 constitutes the first compartment. A passageway 36 having a non-return valve 38 is arranged between the desiccant compartment 34 and the container 26. The non-return valve 38 is arranged whereby air can pass from the inlet 28 through the first compartment 34 and passageway 36 into the container 26 but not in the reverse direction. The desiccant compartment 34 also has a return passageway 40 with a restriction 42 therein. The interior of the container 26 downstream of the desiccant compartment 34 constitutes the second compartment 50 which is the air reservoir of the system. The outlet from the reservoir 26 may include a non-return valve (not shown).

The control valve 32 is switchable between an inlet path and an exhaust path. The inlet path permits air received from pipe 28 to pass into the container 26 via desiccant compartment 34. The exhaust path permits air to flow from compartment 34 to vent 44.

Figs. 2a and 2b illustrate schematically the air flow through the valve and passageway arrangement shown in Figure 2. Fig. 2a shows the air flow when the control valve is switched to the inlet path with air being dried by the desiccant and Fig. 2b shows the air flow when the control valve is switched to the exhaust path with the desiccant being regenerated.

In Figure 2a an air flow A is illustrated flowing from the inlet pipe 28 through the control valve 32 and into the desiccant compartment 34. The air is dried by the desiccant

in the compartment and split into two air flows A1 and A2. Air flow A1 has a much greater mass flow rate than air flow A2 as the air flow A1 is relatively unrestricted while A2 passes through the restrictor 42. Thus dry air accumulates
5 in the container 26 for ultimate use in the air braking system.

Figure 2b shows an air flow B when the component is in a desiccant regeneration state. In such a case, the control valve 32 is switched, for example by a timed control signal,
10 from the inlet pipe 28 to the exhaust vent 44. The air in the container 26 is at a relatively high pressure in relation to ambient atmospheric pressure. Thus dry air B travels via passageway 40 and restrictor 42, through the desiccant compartment 34 and via control valve 32 to the
15 vent 44. The non-return valve 38 prevents reverse flow via passageway 36.

Thus, the component is used to dry air from the inlet 28 and, when the compressor is off load, the control valve 32 can be switched to the vent 44 to allow some dry air from
20 the container 26 to pass through the desiccant compartment 34 for regeneration of the desiccant therein. The restrictor 42 ensures controlled expansion of the pressurised air and effective regeneration. The size of the restrictor is selected to suit the regeneration parameters
25 and is dependent on pressure, saturation volume and other factors which are governed by the system specification.

Figure 3 shows an alternative form of a component in

accordance with the invention. Parts in Figure 3 corresponding to parts in Figure 2 carry the same reference numerals.

The component of Figure 3 is substantially similar to that shown in Figure 2 except two desiccant compartments 34a, 34b with respective passages 36a, 36b and 40a, 40b are provided. As shown in Figs. 3a and 3b the control valve 32 switches the air supply from inlet pipe 28 between the desiccant compartments 34a and 34b. The desiccant compartment 34a, 34b which is not being supplied with air from the inlet 28 can be connected to the exhaust vent 44. Thus, whilst one desiccant compartment dries the air being supplied to the container 26, part of the dry air in container 26 can be vented to atmosphere via the other desiccant compartment to dry the desiccant in that compartment.

The component of Figure 3 is particularly useful where available off-load time for the system is limited.

Figure 3a shows the air flow through the system when desiccant compartment 34a is vented to atmosphere. Air flows into the control valve 32 and is directed into the desiccant compartment 34b. The air flow splits into two air flows B1 and B2, as described previously in relation to Fig. 2, B1 travels along passageway 36b via the non-return valve 38b and B2 travels along the passageway 40b via the restrictor 42b. As the air in container 26 is pressurized relative to the ambient atmospheric pressure, an air flow A

can be arranged to pass via the restrictor 42a in passageway 40a, through the desiccant compartment 34a to dry the desiccant and out via the exhaust vent 44 to atmosphere.

5 Fig. 3b illustrates the system when the desiccant compartment 34b is vented to atmosphere. In that case the air flow travels through the control valve 32 into desiccant compartment 34a and splits into air flows A1, A2. A1 travels, as described previously in relation to Fig. 2, along the passageway 36a via the non-return valve 38a and A2 travels along passageway 40a via the restrictor 42a. Again, 10 the high pressure in container 26 is available to cause an air flow B to travel via passageway 40b through the restrictor 42b and desiccant compartment 34b, drying the desiccant. The moist air in air flow B is then vented to 15 atmosphere via the exhaust vent 44.

It is clear that the system of Figure 3 can be run continuously. When the desiccant bed in one of the compartments 34a, 34b becomes saturated or after a certain time period, the control valve 32 can switch so that the 20 incoming air passes through the dry desiccant bed and the relatively wet desiccant bed is vented to atmosphere for regeneration. The control valve 32 may direct the air between the desiccant beds according to a timer, a sensing device adapted to monitor the degree of moisture in the 25 desiccant beds, or any other suitable sensor. Again the size of the restrictor is dependent on system parameters but it does not matter that air flows through the restrictor in

both directions. What is important is that the majority of flow is via the non-return valve in the drying direction whilst all of the flow is via the restrictor in the regeneration direction.

5 Referring now to Figure 4 there is shown a longitudinal cross-section of a third component 24 according to the present invention. The component 24 comprises an airtight container 26 having an inlet 28 and an outlet 30. The inlet is closed by a control valve 32. The outlet 30 is
10 closable in use by a demand valve of an air braking system (not shown) and may incorporate a non-return valve. The airtight container 26 is provided with an internal partition 46 dividing the interior of the airtight container 26 into two chambers; a service chamber 48 and a purge chamber 50.
15 The partition 46 is provided with a non-return valve 52 permitting fluid communication from the purge chamber 50 to the service chamber 48 but not in the reverse direction.

 A desiccant compartment 34 is located downstream of the inlet 28 within the purge chamber 50. The desiccant
20 compartment 34 constitutes a first compartment, the purge chamber 50 a second compartment, and the service chamber 48 a further compartment. A passageway 36 having a non-return valve 48 is arranged between the desiccant compartment 34 and the purge chamber 50. The non-return valve 48 is
25 arranged whereby air can pass from the inlet 28 through the desiccant compartment 34 and into the purge chamber 50 but not in the reverse direction. The desiccant compartment 34

also has a second passageway 40 with a restriction 42 therein. The service chamber 48 downstream of the purge chamber 50 serves as the air reservoir for the air braking system. The outlet 30 from the service chamber 48 may
5 include a non-return valve (not shown). The control valve 32 is switchable between an inlet path and an exhaust path. The inlet path permits air received from the inlet 28 to pass first into the purge chamber 50 via the desiccant compartment 34 and thereafter to the service chamber 48 via
10 the check valve 52. The exhaust path permits air to flow from the desiccant compartment 34 to a vent 44.

Figures 4a and 4b illustrate schematically the air flow through the valve and passageway arrangement shown in Figure 4. Figure 4a shows the air flow when the control
15 valve 32 is switched to the inlet path with the air being dried by the desiccant and Figure 4b shows the air flow when the control valve 32 is switched to the exhaust path with the desiccant being regenerated.

The component of Figure 4 operates in a manner
20 substantially the same as that described above for the air dryer of Figures 2, 2a and 2b with the exception that air within the service chamber 48 cannot flow back through the desiccant compartment 34 when the control valve 32 is switched to the exhaust path.

25 The composite tank of Fig. 4 is readily constructed from cylinders of e.g. sheet metal and domed ends attached thereto by e.g. welding. It has good resistance to pressure

deformation whilst being economical to construct and of compact dimensions.

Figure 5 shows an alternative form of a component in accordance with the invention. Parts in Figure 5 corresponding to parts in Figure 4 carry the same reference numerals.

The component of Figure 5 is substantially similar to that shown in Figure 4 except two desiccant compartments 34a, 34b with respective passages 36a, 36b and 40a, 40b are provided. As shown in Figs. 5a and 5b the control valve 32 switches the air supply from inlet pipe 28 between the desiccant compartments 34a and 34b. The desiccant compartment 34a, 34b which is not being supplied with air from the inlet 28 can be connected to the exhaust vent 44. Thus, whilst one desiccant compartment dries the air being supplied to the container 26, part of the dry air in container 26 can be vented to atmosphere via the other desiccant compartment to dry the desiccant in that compartment.

The component of Figure 5 is particularly useful where available off-load time for the system is limited.

The component of Figure 5, operates in a manner substantially the same as that described above for the component of Figures 3, 3a and 3b with the exception that air within the service chamber 48 cannot flow back through the desiccant compartments 34a, 34b when the control valve is switched to the exhaust paths.

It is clear that the system of Figure 5 can be run continuously. When the desiccant bed in one of the compartments 34a, 34b becomes saturated or after a certain time period, the control valve 32 can switch so that the incoming air passes through the dry desiccant bed and the relatively wet desiccant bed is vented to atmosphere for regeneration. Again the size of the restrictor is dependent on system parameters but it does not matter that air flows through the restrictor in both directions. What is important is that the majority of flow is via the non-return valve in the drying direction whilst all of the flow is via the restrictor in the regeneration direction.

The control valve 32 in any of the foregoing embodiments can be designed to suit either "line unload" or "governor controlled" air braking systems. In "line unload" systems, when the compressor is in the off-load state, the compressor continues to work but the air is vented to atmosphere. In a "governor controlled" system, when the compressor is in the off-load state, the compressor is deactivated and no longer delivers air.

The present system as set out in Figures 2, 3, 4 or 5 provides a combined air dryer/reservoir component with reduced external piping. That, in turn, allows the system to be fitted on smaller vehicles or on vehicles with restricted space in their engine compartment. The reduction in external piping eliminates possible leak sites at the interface between piping and system components. Furthermore

the compact nature of the air dryer/reservoir allows the unit to be satisfactorily tested before installation.

CLAIMS:

1. A component of an air braking system 24 comprising a container 26 having first and second compartments. 34,50, the first compartment 34, having an
5 upstream inlet 28 for connection to a source of air under pressure, a vent 44 for connection to the exterior of the container 26 and a desiccant therein, the second compartment 50 comprising a chamber having a downstream inlet 36 from the first compartment 34 with non-return means 38 therein,
10 a return connection 40 to the first compartment 34 having a fluid restrictor 42 therein, and an outlet 30 for connection to an air braking system.

2. A component 24 as claimed in claim 1 and further including a control valve 32 to regulate the passage of air
15 through said upstream inlet 28 and vent 44.

3. A component 24 as claimed in claim 2 wherein the control valve 32 is mounted to the container 26.

4. A component 24 as claimed in any preceding claim and further including a third compartment 34b having an
20 upstream inlet 28 for connection to a source of air under pressure, a vent 44 for connection to the exterior of the container 26 and a desiccant therein, the second compartment 50 also having a downstream inlet 36b from the third compartment 34b with non-return means 38b therein, and a
25 return connection 40b to the third compartment 34b having a fluid restrictor 42b therein.

5. A component 24 as claimed in claim 4 and further including a control valve 32 to regulate the passage of air through the upstream inlet 28 and vent 44 of said third compartment 34b.

5 6. A component 24 according to claim 5 wherein a common control valve 32 is provided to regulate the passage of air through said upstream inlets and vents of said first and third compartments 34a, 34b, the control valve being adapted to selectively direct air to the upstream inlet of
10 the first compartment 34a and to the upstream inlet of the third compartment 34b.

7. A component 24 as claimed in claim 6 wherein the control valve 32 is adapted to direct air between said upstream inlets according to a timer.

15 8. A component 24 as claimed in claim 6 wherein the control valve 32 is adapted to direct air between said upstream inlets according to a sensing device adapted to monitor the degree of moisture in said first and third compartments.

20 9. A component 24 as claimed in any preceding claim wherein the compartments 34, 34a, 34b, 50 are defined by a partitions 46 within the container 26.

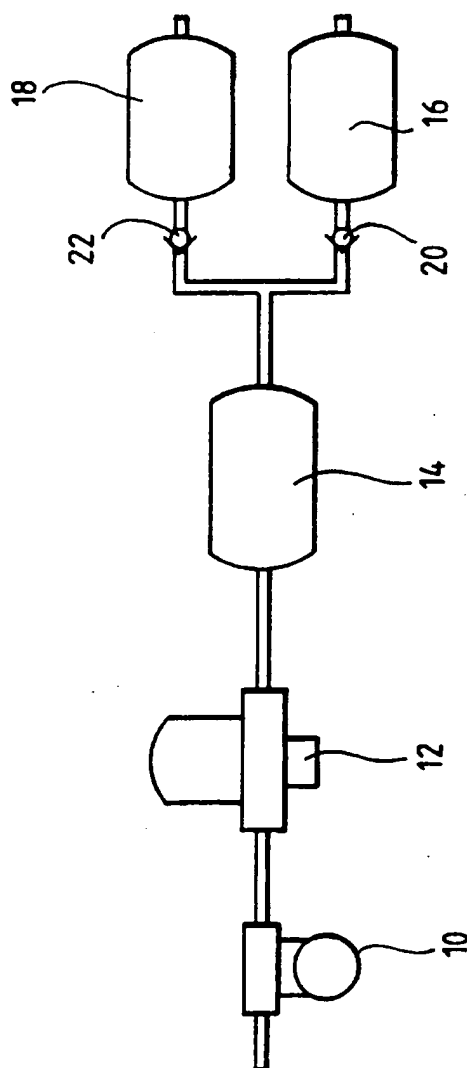
10. A component 24 as claimed in any preceding claim wherein the first compartment 34, 34a projects into the
25 second compartment 50.

11. A component 24 according to any preceding claim and including a further compartment 48 comprising a chamber

connected to said outlet 30, non-return means 52 being provided at said connection and said further compartment 48 including a supply port for connection to an air braking system.

5 12. A component 24 as claimed in claim 12 wherein all of said compartments 34,34a,34b,48.50 are defined by partitions 46 within the container 26.

 13. A vehicle air braking system including a component 24 as claimed in any preceding claim.



PRIOR ART

Fig.1.

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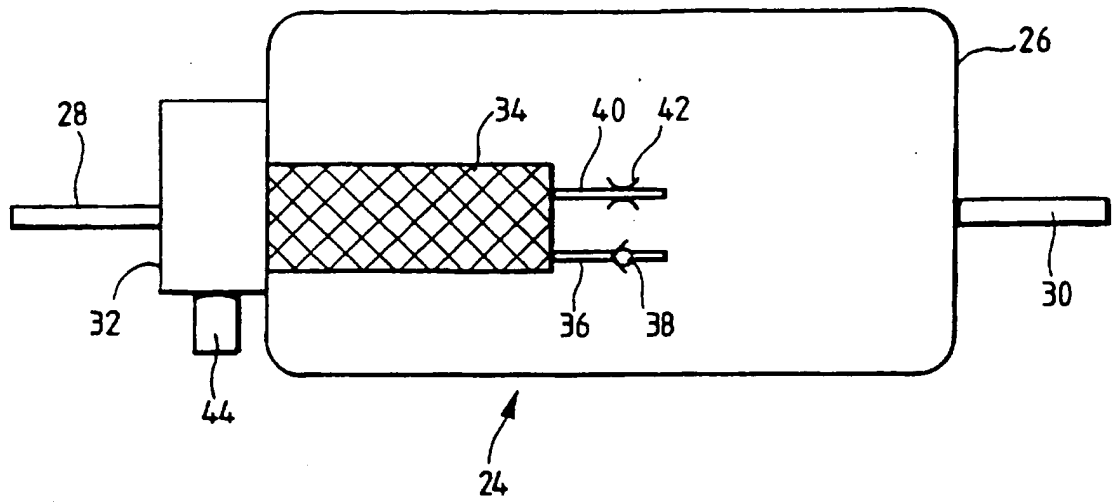


Fig. 2.

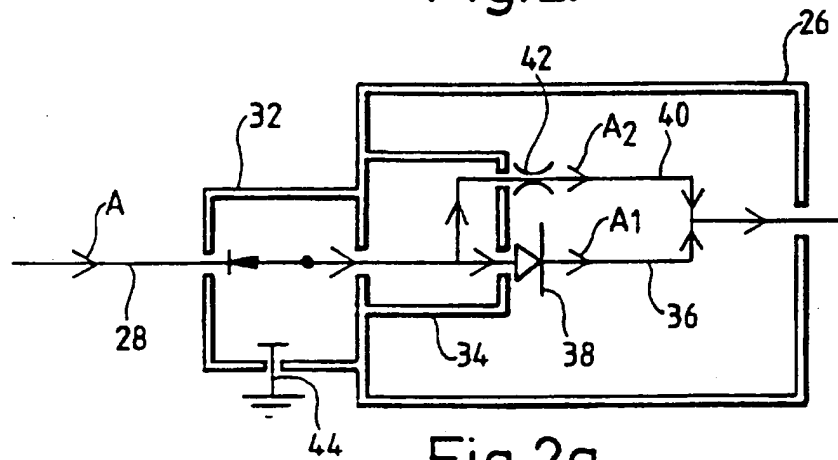


Fig. 2a.

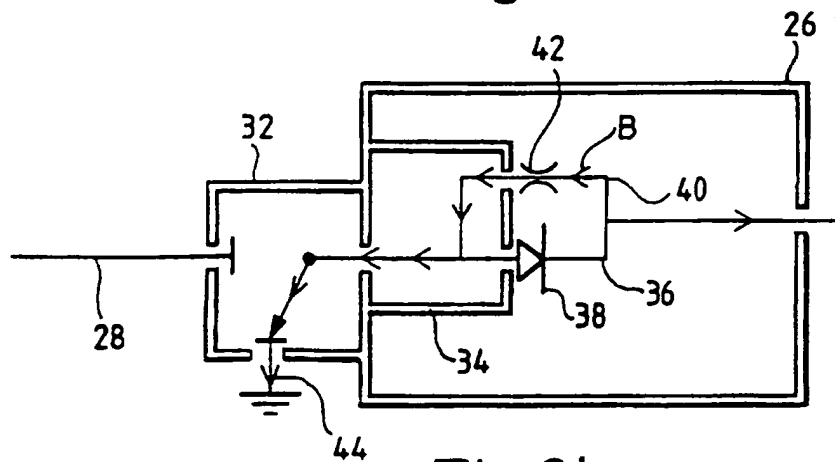


Fig. 2b.

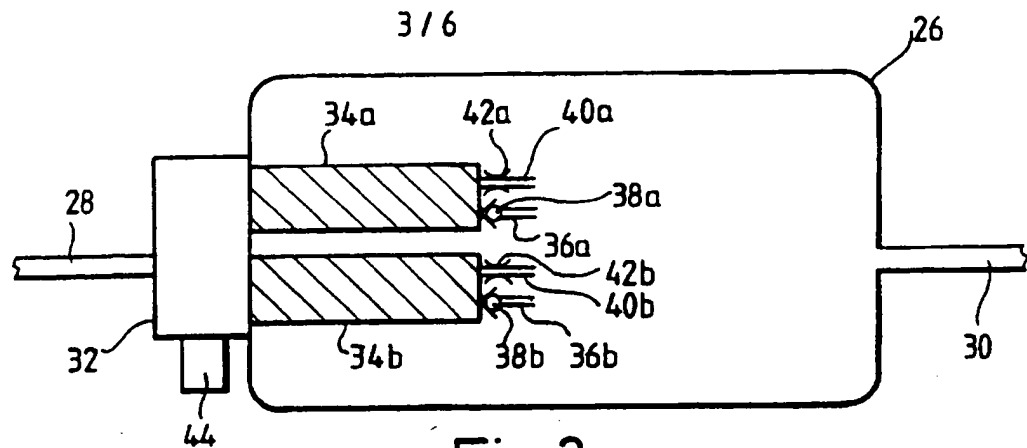


Fig.3.

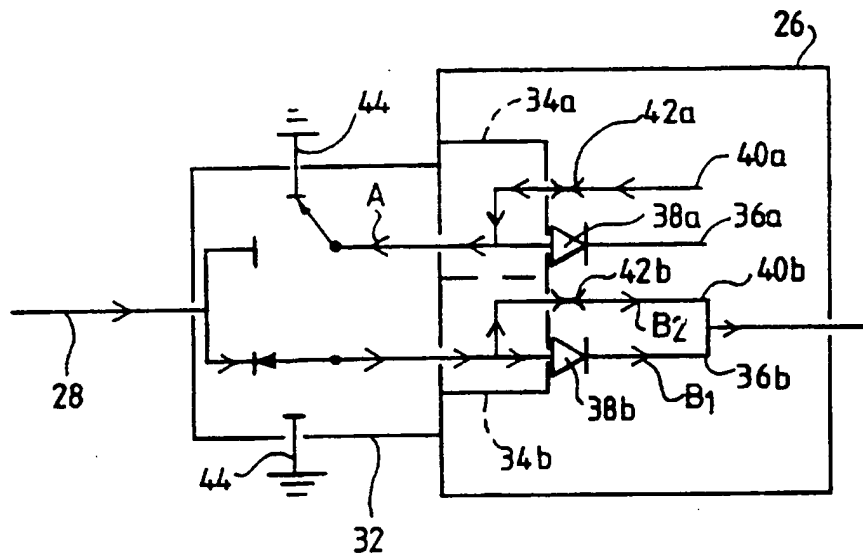


Fig.3a.

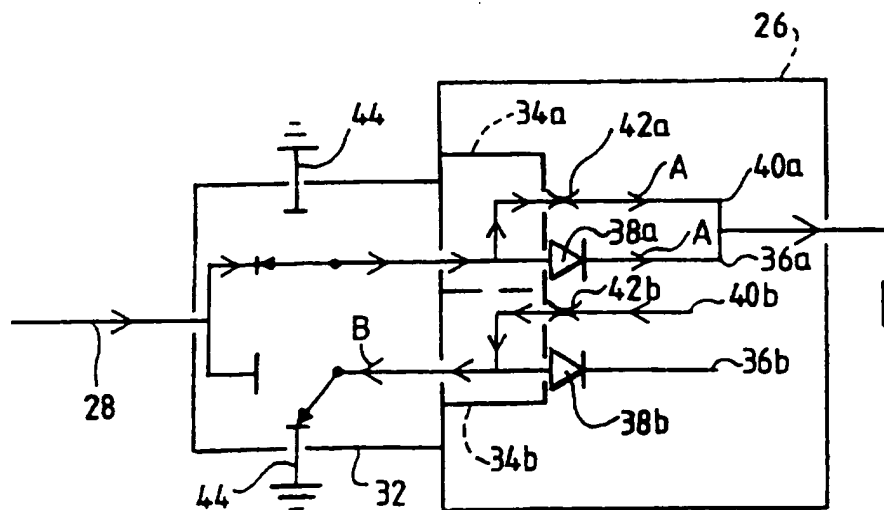


Fig.3b.

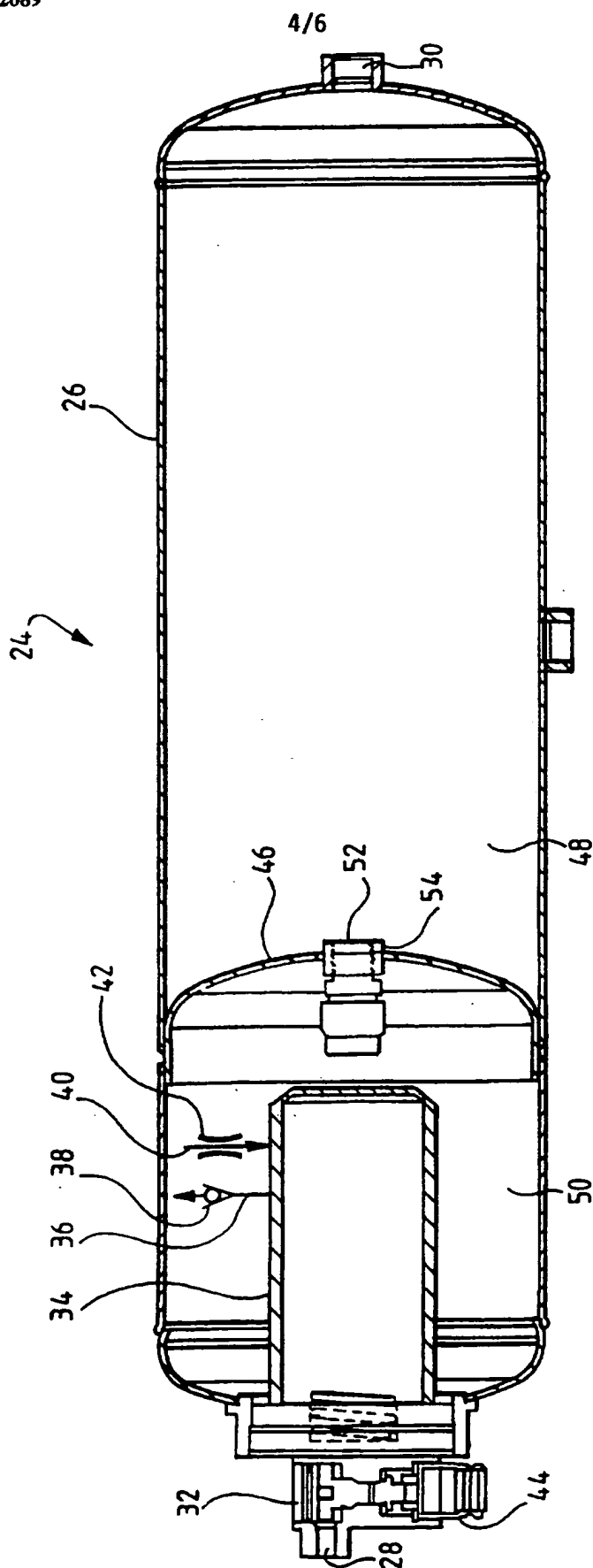


Fig.4.

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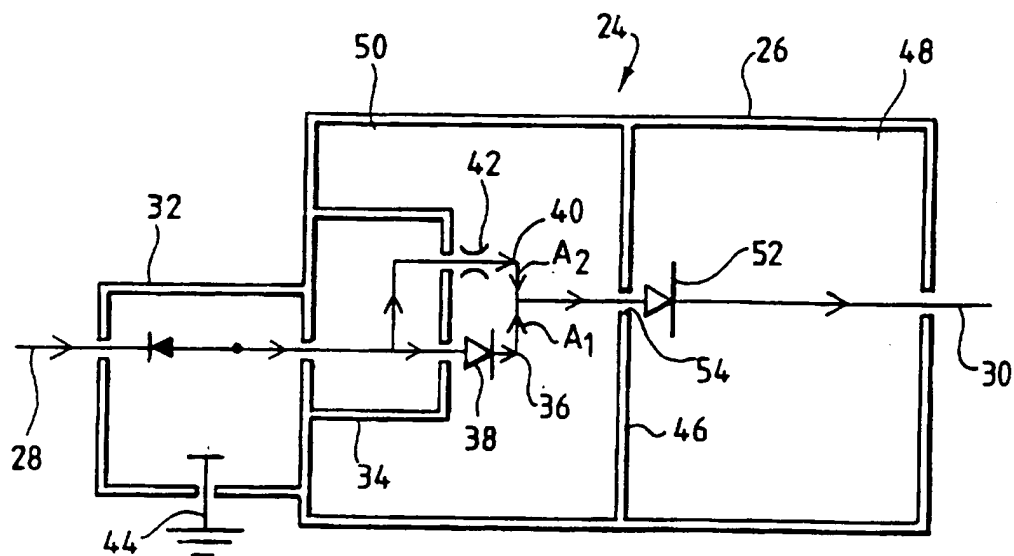


Fig.4a.

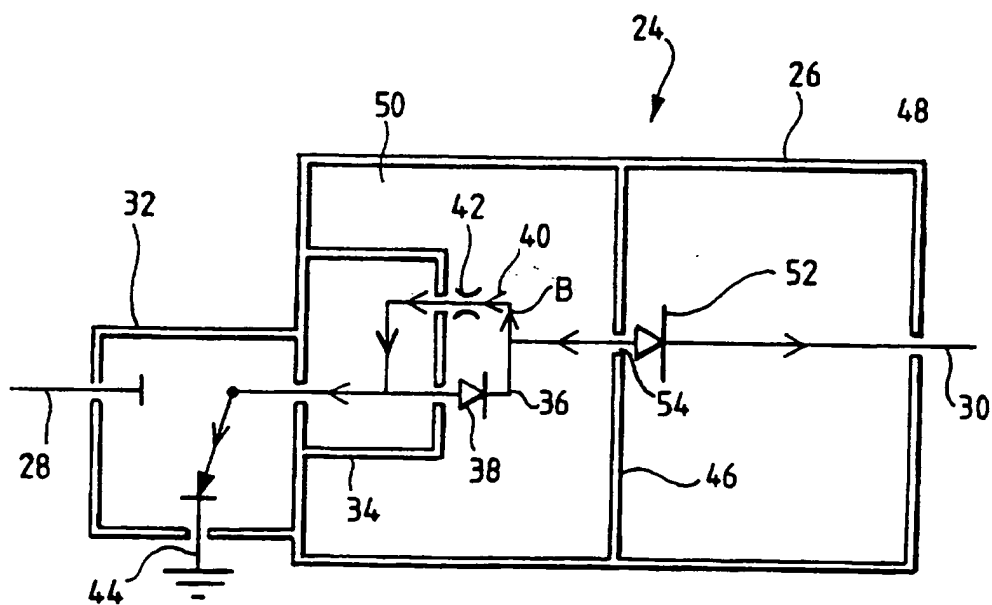


Fig.4b.

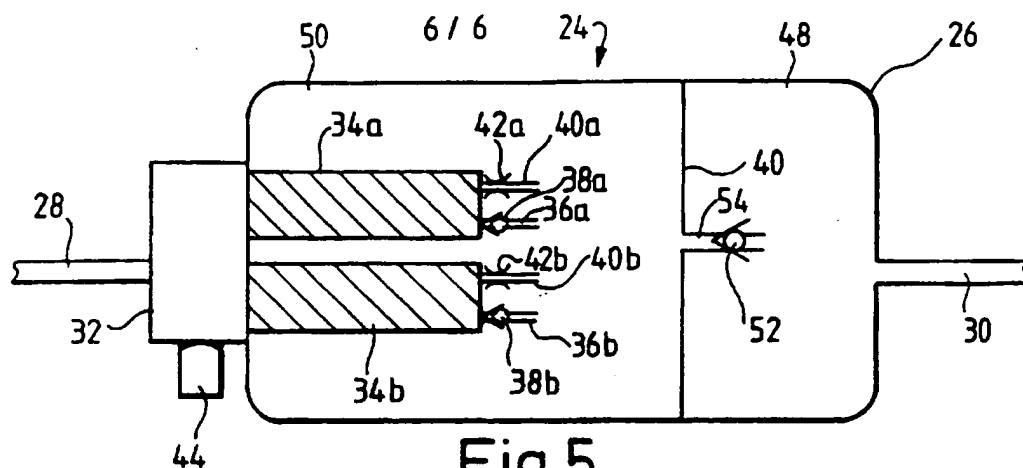


Fig.5.

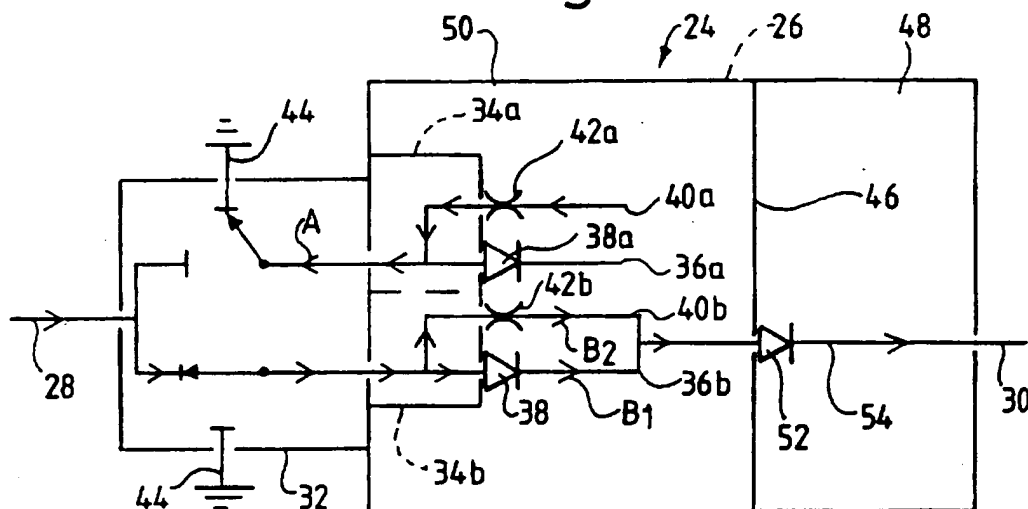


Fig.5a.

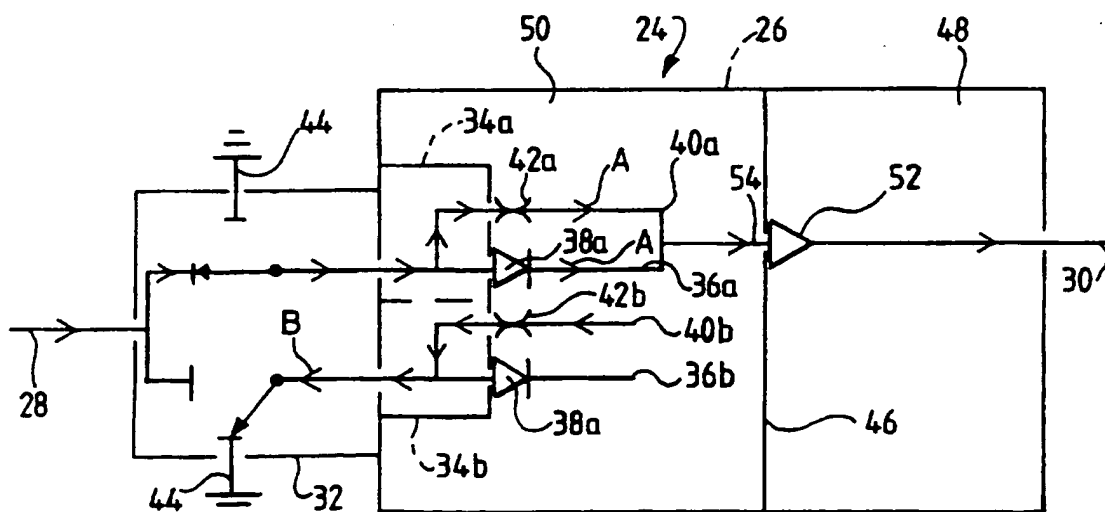


Fig.5b.

INTERNATIONAL SEARCH REPORT

International Application No

PCT/GB 97/02528

A. CLASSIFICATION OF SUBJECT MATTER
IPC 6 B60T17/00

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 6 B60T B01D

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 5 286 283 A (DAVID J. GOODELL) 15 February 1994 see column 3, line 19 - column 4, line 5; figure 1 ----	1-3
A	US 3 714 763 A (KENICHIRO SUZUKI) 6 February 1973 see the whole document ----	1-3
A	US 5 423 129 A (T. KEVIN CASTLE) 13 June 1995 see column 2, line 61 - column 3, line 33; figure 1 ----	1
A	GB 2 163 366 A (NIPPON AIR BRAKE CO) 26 February 1986 see page 1, line 105 - page 2, column 122; figures 1-4 -----	1, 4

☐ Further documents are listed in the continuation of box C.

☒ Patent family members are listed in annex.

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Date of the actual completion of the international search

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European Patent Office, P.B. 5818 Patentlaan 2
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Fax: (+31-70) 340-3018

Authorized officer

Harteveld, C

INTERNATIONAL SEARCH REPORT

Information on patent family members

International Application No

PCT/GB 97/02528

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
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US 3714763 A	06-02-73	DE 2106326 A FR 2081005 A	02-09-71 26-11-71
US 5423129 A	13-06-95	NONE	
GB 2163366 A	26-02-86	US 4655801 A	07-04-87

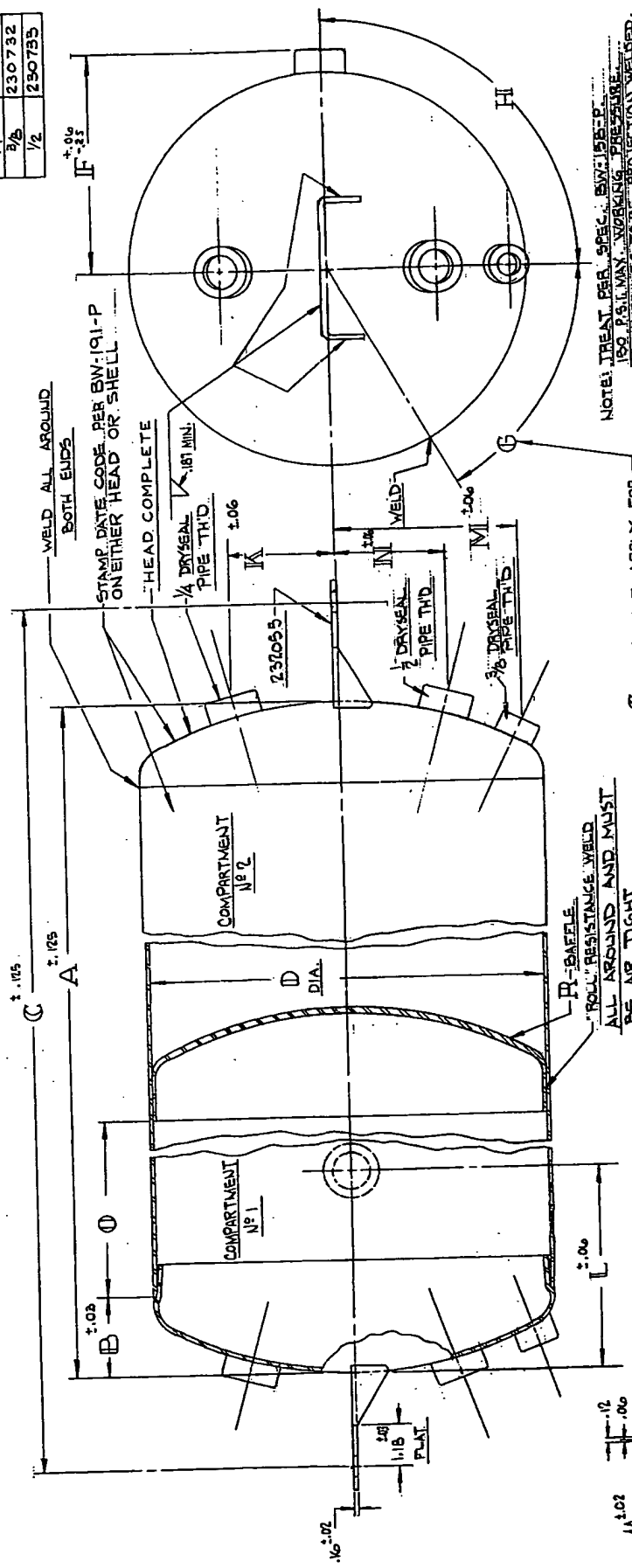
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3/4 DIA. RESERVOIR C12.00 D 9 1/2										
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121	122	249647	1/2	90°	2750	5.34	1/2	90°	42750	5.34
122	123	249648	1/2	90°	2775	5.34	1/2	90°	43175	5.34
123	124	249649	1/2	90°	2800	5.34	1/2	90°	43600	5.34
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134	135	249660	1/2	90°	3075	5.34	1/2	90°	48275	5.34
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143	144	249669	1/2	90°	3300	5.34	1/2	90°	52100	5.34
144	145	249670	1/2	90°	3325	5.34	1/2	90°	52525	5.34
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148	149	249674	1/2	90°	3425	5.34	1/2	90°	54225	5.34
149	150	249675	1/2	90°	3450	5				

[illegible]

① OBSOLETE

PIPE TH'D. SIZE	FERRULE PIECE NO. SHELL
1/4	
3/8	230732
1/2	230733



NOTE: THREAD PER SPEC. BSW: 155-P
ISO P.S.I. MAY, WORKING PRESSURE.
ALL FERRULES TO BE PROTECTION WELDED.
ALL PIPE THREADS MUST BE TO GAGE AND
FITTED WITH THREAD PROTECTIONS.

(G) DOES NOT APPLY FOR -
OPTIONAL HEAD CONSTRUCTION
NOTE: RESERVOIR COMPLIES WITH SAE J-1041
SAE J-1041 OFFENVS 121.

MATERIAL	BRN	APPROVED: R.D. Red	11/9/71
TREATMENT	WF	APPROVED: 11-3-71	11-10-71
		CHECKED: [Signature]	11-10-71

MATERIAL	TREATMENT

2.26 REF. 4.12

BRACKET TYPE
TWO COMPARTMENT RESERVOIR

DRAWN	APPROVED:	R. J. Lutz	11/9/71
WJF			
11-13-71	APPROVED:	J. W. Lutz	11-10-71
	CHECKED		
	APPROVED:		

MATERIAL	TREATMENT

2.26 REF. 4.12

285384

31370580